# What's New in **Ansys** LS-DYNA R16

**Galal Mohamed** 

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#### Release management

Every alternative year LS-DYNA release will get long-term support (LTS) e.g., 14, 16, 18...

Existing versions used in production will continue to get LTS support until discontinued.

Ansys continue to release new version with new features every year.

Upcoming & latest releases						
LS-DYNA 16.0	LTS release	April 2025				
LS-DYNA 14.2	LTS release	June 2025				
LS-DYNA 12.2.2*	LTS release	August 2024				

\* Becoming more widely used for production work (explicit)



### Ansys LS-DYNA

- Tightly coupled, scalable multi-physics solver
- One Code, One Model, Multiphysics



## R16 highlights

• Major updates and enhancements in the following areas:





#### Airbag methods





### History of airbag methods

- CV (Control Volume)
  - Simple pressure, too simple for deployment phase
- ALE (Arbitrary Lagrangian Eulerian)
  - Higher accuracy, but CPU intense and difficult to handle
- CPM (Corpuscular Particle Method)
  - Considers the effect of transient gas dynamics and thermodynamics by using a particle to represent a set of finite air or gas molecules
  - Current state-of-the-art: fast and robust
  - Hits accuracy limits for complex gas flows as observed in curtain airbags, for instance

Need for a more sophisticated approach





### Introducing CPG: A new CFD approach to airbag deployment

- Continuum-based Particle Gas (CPG): Particle method solving Navier-Stokes equations
- Innovative CFD method particularly well-suited for airbag deployment simulations
- Superior accuracy for complex gas flows, as seen in curtain airbags
- Validation studies with multiple partners
- Brand new solver first release R16
- New keyword \*AIRBAG\_CPG





### Introducing CPG: A new CFD approach to airbag deployment

- On-going collaboration with Toyoda Gosei Co. Ltd to validate method and implementation
- Custom-made experimental setup to accelerate development
- The goal is to reduce the need for numerical tuning; this requires:
  - Good fabric material characterisation
  - Good measurement of the inflator characteristics



Experimental data courtesy of Toyoda Gosei Co., Ltd. Numerical model courtesy of JSOL Corporation.



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#### Introducing CPG: real world deployments



Bringing airbag deployment simulations to new levels of predictive accuracy.

**D3PLOT** 

D3PLOT v22 supports the new binary d3dat file, which contains all CPG results data

Model and experimental testing courtesy of JSOL Corporation.

#### ...CPG applications for battery modelling



#### Battery abuse simulations: from single cell to EVs



#### Multi-physics battery modelling – cell short circuit

#### Classic approach EM+Thermal



*Multiphysics* approach: EM+Thermal+Structural+CPG/CPM for venting analysis



#### Contacts



#### Beam elements in segment-to-segment contacts

#### • SOFT = 2

- Has checked contact between shell, solid and thick shell elements for the past 23 years
- New in R16 is the addition of beam and discrete beam elements
  - Checks beam-to-beam, beam-to-shell, beam-to-solid, and beam-to-thick shell
  - Works with all keyword options that segment-to-segment contact supports
  - The interface must include edge-to-edge checking:
    DEPT = 5, DEPTH = 15, DEPTH = 35, DEPTH = 55
  - Beams are assumed to have rectangular cross-sections



Solid impactor penetrates woven fabric modelled by beam elements



#### Thermal contact enhancements

- Single surface contact definition made available for simplified pre-processing
  - Consider example of thermal runaway in battery packs
  - Instead of ~100 surface-to-surface contacts, one single surface contact will suffice
- Thermal edge contact available
  - Works for single surface as well







#### Material models





### **Basic Incremental Failure (BIF)**

- \*MAT\_ADD\_BASIC\_INCREMENTAL\_FAILURE
- Easy to use incremental failure model
  - Require only tensile test for calibration
  - Can achieve good failure modelling when material data is scarce.
  - It works for shell and solid elements (triaxiality and Lode parameter dependence)







### Glass strength prediction model

- Available as \*MAT\_GLASS\_SPM
- Capabilities of the new method
  - Can describe the probabilistic fracture behaviour of glass via sub-critical crack growth (SCG).
  - Predicts the strength of glass plates of various geometries exposed many different load cases.
  - User can select a representative case of a glass fracture strength simply by altering the failure percentile parameter.
  - Example: windshield impact.

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Model details and calibration procedure described in Rudshaug et al., A physically based strength prediction for glass, International LS-DYNA Conference, 2024.

nsys



Force [N]





### MAT\_RRR\_POLYMER/\*MAT\_317

- New material model for thermoplastics
  - Developed in collaboration with IKEA.
  - Handles impact, stress relaxation, and material recovery.
  - Builds on the rheological network framework, containing three links with springs and dampers.
  - Requires 24 material parameters and a sophisticated parameter fit







#### **Implicit Solver**





#### Implicit mechanics – Modal analysis I

- Lanczos flagship eigensolver: improved shifting logic.
- More robust and faster when computing thousands of modes.
- Example: Honda Accord model (courtesy Arup and NHTSA), 35M DOFs, 3k modes:





R16 is 34% faster

### Implicit mechanics – Modal analysis II

- New eigensolver called Fast Lanczos released in R15 (EIGMTH = 103).
- Approximate eigensolver for computing thousands of eigenmodes for NVH applications.
- R16  $\rightarrow$  improved shifting logic, and memory management
- Example, 21.9M dof elect sedan model:



#### Nosie, Vibration, and Hardness (NVH)



### Including residual vectors

- Higher number of mode shapes for higher accuracy, but higher CPU cost
- Residual mode method in R16 offers high accuracy with limited eigenmodes
- The method extracts an additional set of modes based on loading conditions
- Residual modes are orthogonal to retained eigenmodes
- Step 1: generate eigenmodes and residual vectors
- Step 2: run SSD computation with eigenmodes and residual vectors

#### X-displacement (mm)

Freq (kHz)	50 modes 1236 modes 50 modes + resivence vector		50 modes + residual vector
1	1.055	1.068	1.067
10	4.577e-2	6.65e-2	6.517e-2

#### \*FREQUENCY DOMAIN SSD X-displacement X-displacement Freg = 4.6073 Freg = 0.17379 Contours of X-displacement Contours of X-displacement -1.776e-15 2.722e+01 min=-42.8701, at node# 331 min=-42.8701, at node# 380 -4.287e+00 2.021e+01 max=0. at node# 163 max=27.2167, at node# 473 -8.574e+00 1.320e+01 6.191e+00 -1.286e+01 -8.180e-01 -1.715e+01 -2.144e+01 -7.827e+00 -2.572e+01 -1.484e+01 -2.184e+01 -3.001e+01 -3.430e+01 -2.885e+01 -3.586e+01 -3.858e+01 -4.287e+01 -4.287e+01 d3eigv

Freq = 48.049 Contours of X-displacement

min=-1.82797, at node# 638

max=5.29681, at node# 637

Mode 7

5.297e+00 4.584e+00 3.872e+00 2.447e+00 1.734e+00 1.022e+00 3.095e-01 -4.030e-01 -1.115e+00 -1.828e+00

d3resvec

Mode 1

#### Missing mass correction

- \*FREQUENCY\_DOMAIN\_RESPONSE\_SPECTRUM\_MISSING\_MASS\_CORRECTION
- In general, a mode superposition using a limited number of modes will miss some mass
- For response spectrum analysis, static correction can be made by adding static load response for the missing mass
- Missing mass load is calculated by F(static)- $\Sigma(\text{mode load})$

ISVS

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Max. Y-displacement = 0.138

Max. Y-displacement = 0.197

#### Without missing mass correction

### \*FREQUENCY\_DOMAIN\_RANDOM\_VIBRATION

#### New d3rms file

- Users want to do failure analysis using stress in pre-stressed random vibration
- Total stress is the sum of stress in random vibration and pre-stress
- In the past, the 3-sigma (rms) rule was used.

Insys

New d3rms file includes:

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- State 1: RMS response
- State 2: 3-sigma + pre-stress
- State 3: 3-sigma + |pre-stress|





3-sigma only (max: 102 MPa)

3-sigma + prestress (max: 168 MPa)

### Isogeometric Analysis (IGA)







#### Included features for trimmed solids



#### • \*IGA\_POINT\_UVW

 $\rightarrow$  Define parametric points on the IGA solid surface such that they can be constrained with the FE nodes.

- \*CONSTRAINED\_NODAL\_RIGID\_BODY
- \*CONSTRAINED\_EXTRA\_NODES
- \*CONSTRAINED\_SPR2

courtesy of General Motors



### **IGA:** Subcycling

 Enables subcycling for isogeometric solid elements, see \*CONTROL\_SUBCYCLE\_K\_L and \*CONTROL\_SUBCYCLE\_MASS\_SCALED\_PARTS/PART\_SET.



#### Miscellaneous





#### Simplified electric cable modelling with \*DEFINE\_CABLE

- Simplified creation and analysis of electric cables for modelling wire failure in vehicle crash
  - Automatic creation of cables from beam elements
  - Final cables can be a mix of solids/shells/beams
  - Links cross-section data to each original beam element
- Data available in binout (\*DATABASE\_CABLE):
  - Compression (contact) force
  - Cross-section area
- Data summary in ASCII-file:
  - Time and location for maximum compression force and minimum cross-section area for each cable and for whole model

Cable generation from beam elements



#### data summary

#### PART-WISE DATA:

Beam part=	83990001, sampling fre	quency=	1 cycles		
maxstress=	0.4953E-01, time=	0.5000E+01,	element=	46	
minarea =	0.5891E+02, time=	0.5000E+01,	element=	48	
maxforce =	0.5590E+01, time=	0.4998E+01,	element=	46	
Beam part=	83990002, sampling fre	quency=	10 cycles		
maxstress=	0.7170E-01, time=	0.4991E+01,	element=	6020579	
minarea =	0.8419E+02, time=	0.4997E+01,	element=	6020579	
maxforce =	0.9542E+01, time=	0.4997E+01,	element=	6020579	
Beam part=	83990005, sampling fre	quency=	1 cycles		
maxstress=	0.3333E-03, time=	0.4854E+01,	element=	6020613	
minarea =	0.3247E+03, time=	0.2972E+01,	element=	6020613	
maxforce =	0.0000E+00, time=	0.0000E+00,	element=	Θ	
Beam part=	83990006, sampling fre	quency=	1 cycles		
maxstress=	0.5296E-03, time=	0.2155E+01,	element=	6020623	
minarea =	0.8662E+02, time=	0.2664E+01,	element=	6020623	
maxforce =	0.0000E+00, time=	0.0000E+00,	element=	Θ	
DATA FOR ALL	PARTS:				
maxstress=	0.7170E-01, time=	0.4991E+01,	element=	6020579, part=	83990002
minarea =	0.5891E+02. time=	0.5000E+01.	element=	48. part=	83990001

6020579 parts

83990002

0.9542E+01, time= 0.4997E+01, element=



### Rigid body constraint in arbitrary point (CMO = 2)

- Until now, constraints (applied through |CMO| = 1) and prescribed motion have always been translated to act in the centre of gravity of a rigid body.
- By setting |CMO| = 2, the constraints or prescribed motion of the rigid body can be applied in a user specified coordinate.
- This enhancement is available also for nodal rigid bodies.

#### Simulating the closure of rigid-body truck door





- Simulating the closing of body closures is done with rigid bodies, rotating at the hinges.
- Before R16, the only way to achieve this was using **\*PART\_INERTIA** to set the centre of gravity at the hinges.
- However, this changes the physical properties and will not lead to accurate results.

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We are coming to the end!

### **Current development activities**

#### Modular contact

- New modular, generic and universal framework for contacts
- Goal:
  - Unified implementation of contact algorithms
  - Better performance and scalability harnessing modern CPU & GPU architectures
  - Friction, failure and other options will be available across all contact types
- Current status and performance gains so far:
  - New implementation of existing **SOFT = 0/1** contact (**SOFT = 2** in progress)
  - Same keyword-input. Only difference: enabled by setting SOFT = 10/11
  - Currently supported **\*CONTACT** keywords:
    - AUTOMATIC\_SINGLE\_SURFACE
    - AUTOMATIC\_GENERAL[\_INTERIOR][\_EDGE\_ONLY]
    - AUTOMATIC\_SURFACE\_TO\_SURFACE
    - AUTOMATIC\_ONE\_WAY\_SURFACE\_TO\_SURFACE
    - AUTOMATIC\_NODES\_TO\_SURFACE



Car-2-car crash. ~10M elements per car Contact time reduced by **40%** 

- Still in development (mostly available in DEV, limited in R16).
- Name and input format likely to change
- Looking for early feedback!

## **Introducing Ansys Hans**

- High-fidelity human body model
- Exclusively tuned to work with LS-DYNA
- Available now
- Provided with pre-configured PRIMER tree file for positioning





# Thank You!

